ECE 171A: Linear Control System Theory Lecture 11: Input/output responses (II)

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Reading materials: Ch 6.3, Ch 9.1

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Input/output responses

Consider a single-input and single output LTI system

$$
\dot{x} = Ax + Bu, \qquad y = Cx + Du,
$$

where $x \in \mathbb{R}^n$, $u \in \mathbb{R}$, and $y \in \mathbb{R}$.

 \triangleright Step input (also known as Heaviside step function)

$$
u(t) = \begin{cases} 0 & \text{if } t \le 0 \\ 1 & \text{if } t > 0 \end{cases}
$$

 \blacktriangleright Impulse input (also known as delta function)

$$
u(t) = p_{\epsilon}(t) = \begin{cases} 0 & \text{if } t < 0 \\ 1/\epsilon & \text{if } 0 \le t < \epsilon \\ 0 & \text{if } t \ge \epsilon \end{cases} \qquad \delta(t) = \lim_{\epsilon \to 0} p_{\epsilon}(t)
$$

 \triangleright Frequency input (also known as sinusoidal excitation)

$$
u(t) = \sin(\omega t + \phi).
$$

Step response

Figure: Sample step response. The rise time, overshoot, settling time, and steady-state value give the key performance properties of the signal.

- ▶ Steady-state value $y_{\rm ss}$: final level of the output, assuming it converges
- ▶ Rise time T_r : time required for the signal to first go from 10% of its final value to 90% of its final value.
- **Overshoot** M_{p} : the percentage of the final value by which the signal initially rises above the final value
- ▶ Settling time T_s : time required for the signal to stay within 2% of its final value for all future times

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Impulse response

Consider two LTI systems

▶ System 1: open-loop stable system

$$
A_1 = \begin{bmatrix} -1 & 1 \\ 0 & -2 \end{bmatrix}, \quad B = \begin{bmatrix} 0 \\ 1 \end{bmatrix}, \quad C = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad D = 0.
$$

▶ System 2: open-loop unstable system

$$
A_2 = \begin{bmatrix} 1 & 1 \\ 0 & 2 \end{bmatrix}, \quad B = \begin{bmatrix} 0 \\ 1 \end{bmatrix}, \quad C = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad D = 0.
$$

We consider the impulse response

 \blacktriangleright Impulse input (also known as delta function)

$$
u(t) = p_{\epsilon}(t) = \begin{cases} 0 & \text{if } t < 0 \\ 1/\epsilon & \text{if } 0 \le t < \epsilon \\ 0 & \text{if } t \ge \epsilon \end{cases} \qquad \delta(t) = \lim_{\epsilon \to 0} p_{\epsilon}(t)
$$

▶ In the simulation, we choose $\epsilon = 0.01$ seconds.

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▶ Matlab command: sys = ss(A, B, C, D); % create an LTI system \blacktriangleright y = lsim(sys,u,t,x0); % simulate the response to the input u

Case 2: Scale the input $u_2(t) = 2u_1(t) = 2\delta(t)$.

Q: Compared to the response in Case 1, what do you observe? 1 mpulse response $8/24$

Case 3: Shift the input $u_3(t) = u_1(t-1) = \delta(t-1)$.

Q: Compared to the response in Case 1, what do you observe? \blacksquare [Impulse response](#page-4-0) 9/24

Case 4: Shift the input $u_4(t) = u_1(t-2) = \delta(t-2)$.

Q: Compared to the response in Case 1 and Case 3, what do you observe? [Impulse response](#page-4-0) 10/24

Case 5: Sum three inputs $u_5(t) = u_1(t) + u_3(t) + u_4(t)$.

Q: Compared to the response in Cases 1, 3, and 4, what do you observe?

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Input/output response

Impulse response: the output of the system with zero initial condition and having an impulse $\delta(t)$ as its input can be written as

$$
h(t) = \int_0^t Ce^{A(t-\tau)}B\delta(\tau)d\tau + D\delta(t) = Ce^{At}B + D\delta(t).
$$

• Informally: any input $u(t)$ can be decomposed into a sum of impulses

$$
u(t) = \sum_{\tau=0}^{\infty} u(\tau)\delta(t-\tau)
$$

▶ The system response becomes

$$
y(t) = \underbrace{Ce^{At}x(0)}_{\text{Initial response}} + \underbrace{\int_0^t h(t-\tau)u(\tau)d\tau}_{\text{forced response}}
$$

(Forced response: superposition of the response to an infinite set of shifted impulses whose magnitudes are given by the input $u(t)$)

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Frequency response

Frequency input (also known as sinusoidal excitation)

 $u(t) = \sin(\omega t + \phi)$.

Consider three LTI systems

▶ Open-loop stable system 1:

$$
A_1 = \begin{bmatrix} -1 & 4 \\ -3 & -2 \end{bmatrix}
$$
, $B_1 = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$, $C_1 = \begin{bmatrix} 1 & 0 \end{bmatrix}$, $D_1 = 0$.

▶ Open-loop stable system 2:

$$
A_2 = \begin{bmatrix} -1 & 1 \\ 0 & -2 \end{bmatrix}, \quad B_2 = \begin{bmatrix} 0 \\ 5 \end{bmatrix}, \quad C_2 = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad D_2 = 0.
$$

▶ Open-loop unstable system:

$$
A_3 = \begin{bmatrix} 1 & 4 \\ -3 & 2 \end{bmatrix}, \quad B_3 = \begin{bmatrix} 0 \\ 1 \end{bmatrix}, \quad C_3 = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad D_3 = 0.
$$

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Frequency response: unstable system

Frequency input (also known as sinusoidal excitation)

 $u(t) = \sin(\omega t + \phi).$

Take $\omega = 1, \phi = 0$.

Frequency response: stable systems

Frequency response - Bode plot

The steady output has a different amplitude plus a shifted phase.

- \triangleright The gain is the ratio of the amplitudes of the sinusoids, which can be determined by measuring the height of the peaks.
- ▶ The **phase** is determined by comparing the ratio of the time between zero crossings of the input and output.

A convenient way to view the frequency response is to plot how the gain and phase depend on ω — Bode plot

Frequency response - Bode plot

Figure: A frequency response (gain only) computed by measuring the response of individual sinusoids. The figure on the left shows the response of the system as a function of time to a number of different unit magnitude inputs (at different frequencies). The figure on the right shows this same data in a different way, with the magnitude of the response plotted as a function of the input frequency. The filled circles correspond to the particular frequencies shown in the time responses.

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Solution to differential equation

Consider a state-space system

$$
\begin{aligned}\n\dot{x} &= Ax + Bu, \\
y &= Cx + Du\n\end{aligned} \tag{1}
$$

where $x \in \mathbb{R}^n, u \in \mathbb{R}^m, y \in \mathbb{R}^p$.

Theorem The solution to the linear differential equation [\(1\)](#page-19-0) is given by

$$
x(t) = e^{At}x(0) + \int_0^t e^{A(t-\tau)}Bu(\tau)d\tau.
$$

Proof by verification:

 \triangleright Step 1: satisfy the initial condition?

$$
x(0) = e^{A \times 0} x(0) + \int_0^0 e^{A(t-\tau)} B u(\tau) d\tau = I \times x(0).
$$

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Proof

▶ Step 2: satisfy the differential question?

$$
\frac{d}{dt}x(t) = \frac{d}{dt}e^{At}x(0) + \frac{d}{dt}\left(\int_0^t e^{A(t-\tau)}Bu(\tau)d\tau\right)
$$

$$
= Ae^{At}x(0) + \frac{d}{dt}\left(\int_0^t e^{A(t-\tau)}Bu(\tau)d\tau\right)
$$

$$
= Ae^{At}x(0) + \frac{d}{dt}\left(e^{At}\int_0^t e^{-A\tau}Bu(\tau)d\tau\right)
$$
The Product Rule \rightarrow = $He^{At}x(0) + Ae^{At}\int_0^t e^{-A\tau}Bu(\tau)d\tau + e^{At} \times e^{-At}Bu(t)$
$$
= Ae^{At}x(0) + A\int_0^t e^{A(t-\tau)}Bu(\tau)d\tau + Bu(t)
$$

$$
= A\left(e^{At}x(0) + \int_0^t e^{A(t-\tau)}Bu(\tau)d\tau\right) + Bu(t)
$$

$$
= Ax(t) + Bu(t)
$$

 \blacktriangleright This verifies the convolution satisfies the differential equation [\(1\)](#page-19-0).

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The convolution equation

Theorem The solution to the linear differential equation [\(1\)](#page-19-0) is given by

$$
y(t) = Ce^{At}x(0) + \int_0^t Ce^{A(t-\tau)}Bu(\tau)d\tau + Du(t).
$$
 (2)

 \blacktriangleright [\(2\)](#page-21-0) is called the convolution equation.

- \blacktriangleright The output $y(t)$ is *jointly linear* in both the initial conditions $x(0)$ and the input $u(t)$, which follows from the linearity of matrix/vector multiplications and integration.
- ▶ All the linear properties of LTI systems in Lecture 10 can be directly proved from the general solution [\(2\)](#page-21-0).
- \blacktriangleright The dynamics of the system, characterized by A, play an important role in both stability and performance.

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Summary

$$
h(t) = \int_0^t Ce^{A(t-\tau)}B\delta(\tau)d\tau + D\delta(t) = Ce^{At}B + D\delta(t).
$$

▶ Frequency responses

\blacktriangleright The convolution equation

$$
y(t) = Ce^{At}x(0) + \int_0^t Ce^{A(t-\tau)}Bu(\tau)d\tau + Du(t).
$$

another version is $y(t) = \underbrace{Ce^{At}x(0)}_{\text{Initial response}} + \underbrace{\int_0^t h(t-\tau)u(\tau)d\tau}_{\text{forced response}}$

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