ECE 171A: Linear Control System Theory Lecture 11: Input/output responses (II)

Yang Zheng

Assistant Professor, ECE, UCSD

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Reading materials: Ch 6.3, Ch 9.1

Impulse response

Frequency response

The convolution equation

Summary

Impulse response

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Impulse response

Consider two LTI systems

System 1: open-loop stable system

$$A_1 = \begin{bmatrix} -1 & 1\\ 0 & -2 \end{bmatrix}, \quad B = \begin{bmatrix} 0\\ 1 \end{bmatrix}, \quad C = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad D = 0.$$

System 2: open-loop unstable system

$$A_2 = \begin{bmatrix} 1 & 1 \\ 0 & 2 \end{bmatrix}, \quad B = \begin{bmatrix} 0 \\ 1 \end{bmatrix}, \quad C = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad D = 0.$$

We consider the impulse response

Impulse input (also known as delta function)

$$u(t) = p_{\epsilon}(t) = \begin{cases} 0 & \text{if } t < 0\\ 1/\epsilon & \text{if } 0 \le t < \epsilon \\ 0 & \text{if } t \ge \epsilon \end{cases} \qquad \delta(t) = \lim_{\epsilon \to 0} p_{\epsilon}(t)$$

• In the simulation, we choose $\epsilon = 0.01$ seconds.

Impulse response

Matlab command: sys = ss(A, B, C, D); % create an LTI system
 y = lsim(sys,u,t,x0); % simulate the response to the input u



Case 2: Scale the input $u_2(t) = 2u_1(t) = 2\delta(t)$.



 $\ensuremath{\textbf{Q}}\xspace$: Compared to the response in Case 1, what do you observe? Impulse response

Case 3: Shift the input $u_3(t) = u_1(t-1) = \delta(t-1)$.



 $\ensuremath{\textbf{Q}}\xspace$: Compared to the response in Case 1, what do you observe? Impulse response

Case 4: Shift the input $u_4(t) = u_1(t-2) = \delta(t-2)$.



 $\ensuremath{\mathbf{Q}}\xspace$ Compared to the response in Case 1 and Case 3, what do you observe? Impulse response

Case 5: Sum three inputs $u_5(t) = u_1(t) + u_3(t) + u_4(t)$.



Q: Compared to the response in Cases 1, 3, and 4, what do you observe?

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Frequency response

Frequency input (also known as sinusoidal excitation)

 $u(t) = \sin(\omega t + \phi).$

Consider three LTI systems

Open-loop stable system 1:

$$A_1 = \begin{bmatrix} -1 & 4 \\ -3 & -2 \end{bmatrix}, \quad B_1 = \begin{bmatrix} 0 \\ 1 \end{bmatrix}, \quad C_1 = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad D_1 = 0.$$

Open-loop stable system 2:

$$A_2 = \begin{bmatrix} -1 & 1 \\ 0 & -2 \end{bmatrix}, \quad B_2 = \begin{bmatrix} 0 \\ 5 \end{bmatrix}, \quad C_2 = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad D_2 = 0.$$

Open-loop unstable system:

$$A_3 = \begin{bmatrix} 1 & 4 \\ -3 & 2 \end{bmatrix}, \quad B_3 = \begin{bmatrix} 0 \\ 1 \end{bmatrix}, \quad C_3 = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad D_3 = 0.$$

Frequency response

Frequency response: unstable system

Frequency input (also known as sinusoidal excitation)

 $u(t) = \sin(\omega t + \phi).$

Take $\omega = 1, \phi = 0.$



Frequency response: stable systems



Frequency response - Bode plot

The steady output has a different **amplitude** plus a **shifted phase**.

- The gain is the ratio of the amplitudes of the sinusoids, which can be determined by measuring the height of the peaks.
- The phase is determined by comparing the ratio of the time between zero crossings of the input and output.

A convenient way to view the frequency response is to plot how the gain and phase depend on ω — Bode plot



Frequency response - Bode plot



Figure: A frequency response (gain only) computed by measuring the response of individual sinusoids. The figure on the left shows the response of the system as a function of time to a number of different unit magnitude inputs (at different frequencies). The figure on the right shows this same data in a different way, with the magnitude of the response plotted as a function of the input frequency. The filled circles correspond to the particular frequencies shown in the time responses.

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The convolution equation

Solution to differential equation

Consider a state-space system

$$\dot{x} = Ax + Bu,
y = Cx + Du$$
(1)

where $x \in \mathbb{R}^n, u \in \mathbb{R}^m, y \in \mathbb{R}^p$.

Theorem The solution to the linear differential equation (1) is given by

$$x(t) = e^{At}x(0) + \int_0^t e^{A(t-\tau)} Bu(\tau) d\tau.$$

Proof by verification:

Step 1: satisfy the initial condition?

$$x(0) = e^{A \times 0} x(0) + \int_0^0 e^{A(t-\tau)} B u(\tau) d\tau = I \times x(0).$$

The convolution equation

Proof

Step 2: satisfy the differential question?

$$\begin{split} \frac{d}{dt}x(t) &= \frac{d}{dt}e^{At}x(0) + \frac{d}{dt}\left(\int_{0}^{t}e^{A(t-\tau)}Bu(\tau)d\tau\right)\\ &= Ae^{At}x(0) + \frac{d}{dt}\left(\int_{0}^{t}e^{A(t-\tau)}Bu(\tau)d\tau\right)\\ &= Ae^{At}x(0) + \frac{d}{dt}\left(e^{At}\int_{0}^{t}e^{-A\tau}Bu(\tau)d\tau\right)\\ \end{split}$$
Product Rule $\rightarrow = Ae^{At}x(0) + Ae^{At}\int_{0}^{t}e^{-A\tau}Bu(\tau)d\tau + e^{At} \times e^{-At}Bu(t)\\ &= Ae^{At}x(0) + A\int_{0}^{t}e^{A(t-\tau)}Bu(\tau)d\tau + Bu(t)\\ &= A\left(e^{At}x(0) + \int_{0}^{t}e^{A(t-\tau)}Bu(\tau)d\tau\right) + Bu(t)\\ &= Ax(t) + Bu(t) \end{split}$

This verifies the convolution satisfies the differential equation (1).

The convolution equation

The

The convolution equation

Theorem The solution to the linear differential equation (1) is given by

$$y(t) = Ce^{At}x(0) + \int_0^t Ce^{A(t-\tau)}Bu(\tau)d\tau + Du(t).$$
 (2)

(2) is called the convolution equation.

- The output y(t) is jointly linear in both the initial conditions x(0) and the input u(t), which follows from the linearity of matrix/vector multiplications and integration.
- All the linear properties of LTI systems in Lecture 10 can be directly proved from the general solution (2).
- The dynamics of the system, characterized by A, play an important role in both stability and performance.

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$$h(t) = \int_0^t C e^{A(t-\tau)} B\delta(\tau) d\tau + D\delta(t) = C e^{At} B + D\delta(t).$$

Frequency responses



The convolution equation

$$\begin{split} y(t) &= C e^{At} x(0) + \int_0^t C e^{A(t-\tau)} B u(\tau) d\tau + D u(t). \\ \text{another version is } y(t) &= \underbrace{C e^{At} x(0)}_{\text{Initial response}} + \underbrace{\int_0^t h(t-\tau) u(\tau) d\tau}_{\text{forced response}} \end{split}$$

Summary