

ECE 171A: Linear Control System Theory

Lecture 12: Transfer functions (I)

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Outline

Frequency response

Determining transfer functions

Block Diagrams and Transfer Functions

Summary

The convolution equation

Consider a state-space system

$$\begin{aligned}\dot{x} &= Ax + Bu, \\ y &= Cx + Du\end{aligned}\tag{1}$$

where $x \in \mathbb{R}^n$, $u \in \mathbb{R}^m$, $y \in \mathbb{R}^p$.

Theorem

The solution to the linear differential equation (1) is given by

$$y(t) = Ce^{At}x(0) + C \int_0^t e^{A(t-\tau)} Bu(\tau) d\tau + Du(t).\tag{2}$$

Some observations:

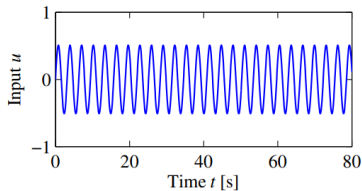
- ▶ Control design is to design an input signal $u(t)$ to shape $y(t)$ (stability, tracking performance, less overshoot, less oscillation, robustness, etc.);
- ▶ The solution (2) is too complex to use for designing a controller;
- ▶ We look for some elegant and simple tools: make the mapping from $u(t)$ to $y(t)$ easier to compute — **Transfer functions**

Steady-state response

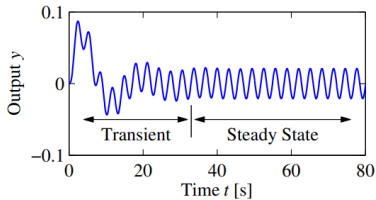
A common practice in evaluating the response of a linear system is to separate out the **short-term response** from the **long-term response**.

- ▶ **Transient response:** which occurs in the first period of time after the input is applied.
It reflects the mismatch between the initial condition and the steady-state solution
- ▶ **Steady-state response:** which is the portion of the output response that reflects the long-term behavior of the system under the given inputs.
 - For inputs that are periodic, the steady-state response will often be periodic (e.g., **frequency response**)
 - For constant inputs, the response will often be constant (e.g., **step response**)

Example



(a) Input



(b) Output

Figure 6.8: Transient versus steady-state response. The input to a linear system is shown in (a), and the corresponding output with $x(0) = 0$ is shown in (b). The output signal initially undergoes a transient before settling into its steady-state behavior.

Step response

Step input (also known as Heaviside step function)

$$u(t) = \begin{cases} 0 & \text{if } t \leq 0 \\ 1 & \text{if } t > 0 \end{cases}$$

- ▶ Let's assume $x(0) = 0$
- ▶ Solve the step response

$$\begin{aligned} y(t) &= Ce^{At}x(0) + \int_0^t Ce^{A(t-\tau)}Bu(\tau)d\tau + Du(t) \\ &= \int_0^t Ce^{At} \times e^{-A\tau}Bd\tau + D = Ce^{At} \int_0^t e^{-A\tau}d\tau B + D \\ &= Ce^{At} \left(-A^{-1}e^{-A\tau} \Big|_{\tau=0}^{\tau=t} \right) B + D \\ &= \underbrace{CA^{-1}e^{At}B}_{\text{transient}} + \underbrace{D - CA^{-1}B}_{\text{steady-state}} \end{aligned}$$

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Frequency response

The **frequency response** of an input/output system measures the way in which the system responds to a sinusoidal excitation.

- ▶ The particular solution associated with a sinusoidal excitation is itself a sinusoid at the same frequency.
- ▶ We can compare the **magnitude** and **phase** of the output sinusoid to the input (— **Transfer function** and **Bode plot**).

Let us consider a sinusoid input

$$u(t) = \cos \omega t.$$

- ▶ Evaluating the convolution equation (2) with input $u(t) = \cos \omega t$ can be very messy.
- ▶ We use the fact that the system is linear to simplify the derivation.
- ▶ In particular, Euler's formula tells us that

$$\cos \omega t = \frac{1}{2} \left(e^{i\omega t} + e^{-i\omega t} \right)$$

- ▶ Thanks to the linearity, we can use the exponential input $u(t) = e^{st}$, and then construct the solution by letting $s = i\omega$ and $s = -i\omega$.

Frequency response - derivation

- ▶ We apply the convolution equation to $u = e^{st}$

$$\begin{aligned}y(t) &= Ce^{At}x(0) + C \int_0^t e^{A(t-\tau)} Be^{s\tau} d\tau + De^{st} \\ &= Ce^{At}x(0) + Ce^{At} \int_0^t e^{(sI-A)\tau} d\tau B + De^{st}\end{aligned}$$

- ▶ We assume $(sI - A)$ is invertible, then

$$\begin{aligned}y(t) &= Ce^{At}x(0) + Ce^{At} \left((sI - A)^{-1} e^{(sI-A)\tau} \Big|_{\tau=0}^{\tau=t} \right) B + De^{st} \\ &= Ce^{At}x(0) + Ce^{At} (sI - A)^{-1} \left(e^{(sI-A)t} - I \right) B + De^{st} \\ &= Ce^{At}x(0) + C(sI - A)^{-1} e^{st} B - Ce^{At} (sI - A)^{-1} B + De^{st}\end{aligned}$$

- ▶ Finally, we obtain

$$y(t) = \underbrace{Ce^{At} (x(0) - (sI - A)^{-1} B)}_{\text{transient}} + \underbrace{(C(sI - A)^{-1} B + D) e^{st}}_{\text{steady-state}}$$

Frequency response - steady-state component

The solution corresponding to the exponential input e^{st} is

$$y(t) = \underbrace{C e^{At} (x(0) - (sI - A)^{-1} B)}_{\text{transient}} + \underbrace{(C(sI - A)^{-1} B + D) e^{st}}_{\text{steady-state}}$$

- ▶ If the system matrix A is stable, the transient component decays to zero
- ▶ **The steady-state component** is proportional to the exponential input e^{st} .
- ▶ We can write the steady-state response as

$$y_{ss}(t) = M e^{i\theta} e^{st} = M e^{st+i\theta},$$

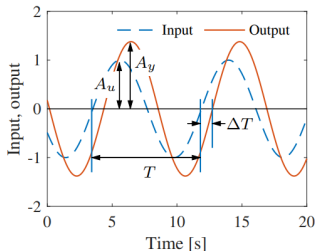
where $G(s) = C(sI - A)^{-1} B + D \leftarrow$ **Transfer function**

- ▶ When $s = i\omega$, we call
 - $M = |G(i\omega)|$ the **gain**, and
 - $\theta = \arg(G(i\omega))$ the **phase** of the system at the forcing frequency ω

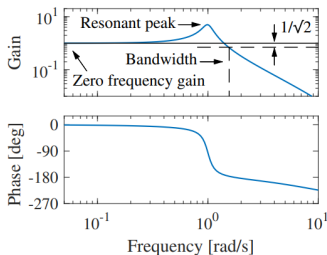
Frequency response - steady-state component

The steady-state solution for a sinusoid $u = \cos \omega t$ is given by

$$y_{ss}(t) = |G(i\omega)| \cos(\omega t + \angle G(i\omega))$$



(a) Input/output response



(b) Frequency response

Figure 6.11: Steady-state response of an asymptotically stable linear system to a sinusoid. (a) A sinusoidal input of magnitude A_u (dashed) gives a sinusoidal output of magnitude A_y (solid), delayed by ΔT seconds. (b) Frequency response, showing gain and phase. The gain is given by the ratio of the output amplitude to the input amplitude, $M = A_y/A_u$. The phase lag is given by $\theta = -2\pi\Delta T/T$; it is negative for the case shown because the output lags the input.

Some terminology

- ▶ **Zero frequency gain:** The gain of a system at ω , corresponds to the ratio between a constant input and the steady output

$$M_0 = G(0) = -CA^{-1}B + D.$$

- The zero frequency gain is well defined only if A is invertible.
 - Zero frequency gain is a relevant quantity only for **stable systems**.
 - In EE, the zero frequency gain is often called the **DC gain**.
- ▶ The **bandwidth** ω_b of a system is the frequency range over which the gain has decreased by no more than a factor $1/\sqrt{2}$ from its reference value (either *zero-frequency gain* or *high-frequency gain*)
- ▶ **Resonant peak** M_r , the largest value of the frequency response
- ▶ **Peak frequency** ω_{mr} , the frequency where the maximum occurs
 - The frequency of the sinusoidal input that produces the largest possible output and the gain at the frequency.

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Transfer functions

Transfer functions: transmission of exponential signals e^{st} with $s = \sigma + i\omega$

$$e^{st} = e^{\sigma t} e^{i\omega t} = e^{\sigma t} (\cos \omega t + i \sin \omega t)$$

where $\sigma \leq 0$: decay rate.

- ▶ Find the transfer function for the state-space system

$$\dot{x} = Ax + Bu, \quad y = Cx + Du. \quad (3)$$

- ▶ The output $y(t)$ of system (3) to the input e^{st} is

$$y(t) = \underbrace{C e^{At} (x(0) - (sI - A)^{-1} B)}_{\text{transient}} + \underbrace{(C(sI - A)^{-1} B + D) e^{st}}_{\text{steady-state}}$$

- ▶ The **transfer function** from u to y of the system (3) is the coefficient of the term e^{st} , i.e.,

$$G(s) = C(sI - A)^{-1} B + D.$$

Example: calculating transfer function

Example

Consider an LTI system

$$\begin{aligned}\dot{x}_1 &= -a_1x_1 - a_2x_2 + u \\ \dot{x}_2 &= x_1\end{aligned}\quad y = x_2$$

- ▶ The system matrices are

$$A = \begin{bmatrix} -a_1 & -a_2 \\ 1 & 0 \end{bmatrix}, B = \begin{bmatrix} 1 \\ 0 \end{bmatrix}, C = [0 \quad 1], D = 0.$$

- ▶ Compute its transfer function

$$\begin{aligned}G(s) &= C(sI - A)^{-1}B + D = [0 \quad 1] \begin{bmatrix} s + a_1 & a_2 \\ -1 & s \end{bmatrix}^{-1} \begin{bmatrix} 1 \\ 0 \end{bmatrix} \\ &= [0 \quad 1] \frac{1}{s^2 + a_1s + a_2} \begin{bmatrix} s & -a_2 \\ 1 & s + a_1 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix} \\ &= \frac{1}{s^2 + a_1s + a_2}.\end{aligned}$$

Example: computing steady-state responses

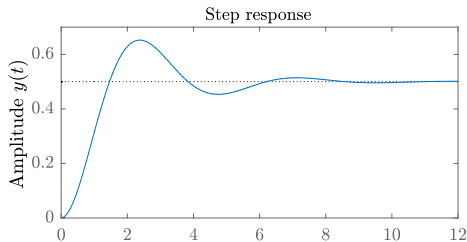
Example

- Suppose $a_1 = 1, a_2 = 2$. Its transfer function is

$$G(s) = \frac{1}{s^2 + s + 2}.$$

- The steady-state response to a step input $u(t) = 1$ is e^{st} with $s = 0$, i.e.

$$y_{ss} = G(0)u = \frac{1}{2}.$$



Example: computing steady-state responses

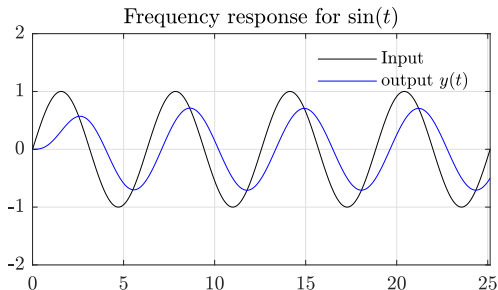
Example

- ▶ The steady-state response to a sin input $u(t) = \sin \omega t$ is

$$y = M \sin(\omega t + \theta), \quad \text{where } M = |G(i\omega)|, \theta = \arg(G(i\omega))$$

- ▶ **Case 1:** $u(t) = \sin t \rightarrow y(t)$?

$$G(i\omega) = \frac{1}{(i\omega)^2 + i\omega + 2}, \quad M = |G(i)| = \frac{1}{\sqrt{2}}, \quad \theta = -45^\circ$$



Example: computing steady-state responses

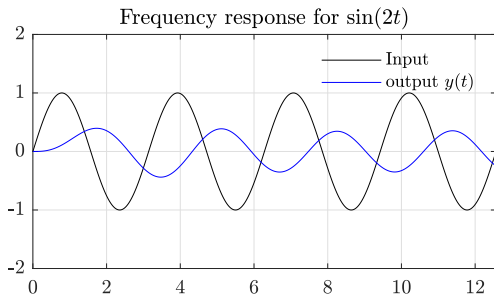
Example

- ▶ The steady-state response to a sin input $u(t) = \sin \omega t$ is

$$y = M \sin(\omega t + \phi), \quad \text{where } M = |G(i\omega)|, \theta = \arg(G(i\omega))$$

- ▶ **Case 2:** $u(t) = \sin 2t \rightarrow y(t)$?

$$G(i2) = \frac{1}{-2 + i2}, \quad M = |G(i2)| = \frac{1}{2\sqrt{2}}, \quad \theta = -135^\circ$$



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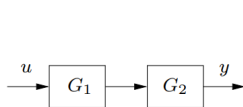
Block Diagrams and Transfer Functions

Summary

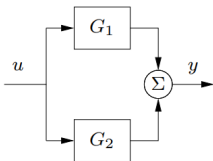
Block diagrams

The combination of block diagrams and transfer functions is a powerful way to represent control systems.

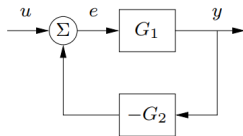
- ▶ Input-output relationship can be derived by **algebraic manipulations** of the transfer functions.



(a) $G_{yu} = G_2G_1$



(b) $G_{yu} = G_1 + G_2$



(c) $G_{yu} = \frac{G_1}{1 + G_1G_2}$

Figure: Interconnections of linear systems. **Series** (a), **parallel** (b), and **feedback** (c) connections are shown.

Feedback connection

- ▶ It is easy to see the relationship

$$y = G_1 e, \quad e = u - G_2 y$$

- ▶ Elimination of e gives

$$\begin{aligned} y = G_1(u - G_2 y) &\Rightarrow (1 + G_1 G_2)y = G_1 u \\ &\Rightarrow y = \frac{G_1}{1 + G_1 G_2} u \end{aligned}$$

- ▶ The transfer function of the feedback connection is thus

$$G = \frac{G_1}{1 + G_1 G_2}.$$

These three basic interconnections can be used as the basis for computing transfer functions for more complicated systems.

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- ▶ Transient response and steady-state response

$$y(t) = \underbrace{C e^{At} (x(0) - (sI - A)^{-1} B)}_{\text{transient}} + \underbrace{(C(sI - A)^{-1} B + D) e^{st}}_{\text{steady-state}}$$

- ▶ Transfer function

$$G(s) = C(sI - A)^{-1} B + D.$$

- Express the steady-state solution of a stable linear system forced by a sinusoidal input

$$u(t) = \sin(\omega t) \rightarrow y_{ss} = |G(i\omega)| \sin(\omega t + \angle G(i\omega))$$

- ▶ **Frequency domain modeling:** Modeling a system through its response to sinusoidal and exponential signals.
 - We represent the dynamics of the system in terms of the generalized frequency s rather than the time domain variable t .
 - The **transfer function** provides a complete representation of a linear system in the frequency domain.