#### **Optimal control and Convex optimization**

## Lecture 1: Problem formulation

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**Disclaimer**: These notes have not been subjected to the usual scrutiny reserved for formal publications. They were developed when the author was a postdoc in Prof. Na Li's group at Harvard. Any typos should be sent to *zhengy@eng.ucsd.edu*.

#### Learning goals:

- 1. Problem formulation of optimal control;
- 2. State-space model of the closed-loop system;
- 3. Well-posedness of feedback systems;
- 4. Internal stability.

# 1 Problem formulation

We consider continuous-time linear time-invariant (LTI) systems of the form

$$\dot{x} = Ax + B_1 w + B_2 u, 
z = C_1 x + D_{11} w + D_{12} u, 
y = C_2 x + D_{21} w + D_{22} u,$$
(1)

where  $x \in \mathbb{R}^n, u \in \mathbb{R}^m, w \in \mathbb{R}^d, y \in \mathbb{R}^p, z \in \mathbb{R}^q$  are the state vector, control action, external disturbance, measurement, and regulated output, respectively. System (1) can be written as

$$\mathbf{P} = \begin{bmatrix} A & B_1 & B_2 \\ \hline C_1 & D_{11} & D_{12} \\ C_2 & D_{21} & D_{22} \end{bmatrix} = \begin{bmatrix} \mathbf{P}_{11} & \mathbf{P}_{12} \\ \mathbf{P}_{21} & \mathbf{P}_{22} \end{bmatrix}$$

where  $\mathbf{P}_{ij} = C_i(sI - A)^{-1}B_j + D_{ij}$ . We refer to **P** as the open-loop plant model.

Consider a dynamic output feedback controller  $\mathbf{u} = \mathbf{K}\mathbf{y}$ , where **K** has a state-space realization

$$\begin{split} \dot{\xi} &= A_k \xi + B_k y, \\ u &= C_k \xi + D_k y. \end{split}$$

$$(2)$$

,

where  $\xi \in \mathbb{R}^{n_k}$  is the internal state of controller **K**. We have  $\mathbf{K} = C_k(sI - A_k)^{-1}B_k + D_k$ . Figure 1 shows a schematic diagram of the interconnection of plant **P** and controller **K**.

#### Problem formulation: Optimal control

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Informally speaking, we aim to find a controller  $\mathbf{K}$  such that the closed-loop system is internally stable and achieves/minimizes desired performance specification:

$$\begin{array}{ll}
\min_{\mathbf{K}} & f(\mathbf{P}, \mathbf{K}) \\
\text{ubject to} & \mathbf{K} \text{ internally stabilizes } \mathbf{P}.
\end{array}$$
(3)

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Figure 1: Interconnection of the plant  ${\bf P}$  and controller  ${\bf K}$ 

where  $f(\mathbf{P}, \mathbf{K})$  defines a certain performance index.

In (3), the performance index  $f(\mathbf{P}, \mathbf{K})$  quantifies the influence of the disturbance w on the performance output z, which is usually captured by a certain norm  $(\mathcal{H}_2 \text{ or } \mathcal{H}_\infty)$  of the closed-loop response from w to z.

## 1.1 Closed-loop system in the frequency domain

By (1), we have

$$\begin{aligned} \mathbf{z} &= \mathbf{P}_{11}\mathbf{w} + \mathbf{P}_{12}\mathbf{u}, \\ \mathbf{y} &= \mathbf{P}_{21}\mathbf{w} + \mathbf{P}_{22}\mathbf{u}. \end{aligned}$$

Considering the controller  $\mathbf{u} = \mathbf{K}\mathbf{y}$ , some simple algebra leads to

$$\mathbf{z} = (\mathbf{P}_{11} + \mathbf{P}_{12}\mathbf{K}(I - \mathbf{P}_{22}\mathbf{K})^{-1}\mathbf{P}_{21})\mathbf{w}.$$
(4)

Thus, the closed-loop response from  $\mathbf{w}$  to  $\mathbf{z}$  is

$$\mathbf{T}_{zw} = \mathbf{P}_{11} + \mathbf{P}_{12}\mathbf{K}(I - \mathbf{P}_{22}\mathbf{K})^{-1}\mathbf{P}_{21}.$$

In (3), the cost function is typically chosen as

$$f(\mathbf{P}, \mathbf{K}) = \|\mathbf{P}_{11} + \mathbf{P}_{12}\mathbf{K}(I - \mathbf{P}_{22}\mathbf{K})^{-1}\mathbf{P}_{21}\|,$$

where  $\|\cdot\|$  can be chosen as the  $\mathcal{H}_2$  or  $\mathcal{H}_{\infty}$  norm.

In (4), one question to ask is when the inverse  $(I - \mathbf{P}_{22}\mathbf{K})^{-1}$  exists. This is related to the the notion of *well-posedness* of feedback systems, which will be defined later.

### 1.2 Closed-loop system in the state-space domain

We can also derive the closed-loop system in the state-space domain, which is a state-space realization of (4). Combining (1) with (2) leads to

$$\frac{d}{dt} \begin{bmatrix} x\\ \xi \end{bmatrix} = \begin{bmatrix} A & 0\\ 0 & A_k \end{bmatrix} \begin{bmatrix} x\\ \xi \end{bmatrix} + \begin{bmatrix} 0 & B_2\\ B_k & 0 \end{bmatrix} \begin{bmatrix} y\\ u \end{bmatrix} + \begin{bmatrix} B_1\\ 0 \end{bmatrix} w,$$
(5a)

$$\begin{bmatrix} y\\ u \end{bmatrix} = \begin{bmatrix} C_2 & 0\\ 0 & C_k \end{bmatrix} \begin{bmatrix} x\\ \xi \end{bmatrix} + \begin{bmatrix} 0 & D_{22}\\ D_k & 0 \end{bmatrix} \begin{bmatrix} y\\ u \end{bmatrix} + \begin{bmatrix} D_{21}\\ 0 \end{bmatrix} w,$$
(5b)

$$z = \begin{bmatrix} C_1 & 0 \end{bmatrix} \begin{bmatrix} x \\ \xi \end{bmatrix} + \begin{bmatrix} 0 & D_{12} \end{bmatrix} \begin{bmatrix} y \\ u \end{bmatrix} + D_{11}w$$
 (5c)

From (5b), we have

$$\begin{bmatrix} I & -D_{22} \\ -D_k & I \end{bmatrix} \begin{bmatrix} y \\ u \end{bmatrix} = \begin{bmatrix} C_2 & 0 \\ 0 & C_k \end{bmatrix} \begin{bmatrix} x \\ \xi \end{bmatrix} + \begin{bmatrix} D_{21} \\ 0 \end{bmatrix} w,$$
(6)

This equation has a unique solution if and only if the following matrix

$$\begin{bmatrix} I & -D_{22} \\ -D_k & I \end{bmatrix}$$

is invertible, which is equivalent to that  $I - D_{22}D_k$  or  $I - D_kD_{22}$  is invertible<sup>1</sup>. Note that the matrix dimension  $D_{22} \in \mathbb{R}^{p \times m}$  and  $D_k \in \mathbb{R}^{m \times p}$ . This is also the condition of well-posedness (there are other equivalent definitions of well-posedness).

**Definition 1.** A feedback system is said to be well-posed if the solutions u(t) and y(t) are unique, given any initial condition x(0) and  $\xi(0)$  and  $w(t) = 0, \forall t > 0$ .

**Lemma 1.** The feedback system in Figure 1 is well-posed if and only if  $I - D_{22}D_k$  is invertible.

For simplicity, we make the following assumption.

**Assumption 1.** It is assumed that the plant is strictly proper, i.e.  $D_{22} = 0$ . By Lemma 1, this guarantees that the closed-loop system is always well-posed.

Now, (6) becomes

$$\begin{bmatrix} y \\ u \end{bmatrix} = \begin{bmatrix} C_2 & 0 \\ D_k C_2 & C_k \end{bmatrix} \begin{bmatrix} x \\ \xi \end{bmatrix} + \begin{bmatrix} D_{21} \\ D_k D_{21} \end{bmatrix} w$$

Substituting this into (7) leads to

$$\frac{d}{dt} \begin{bmatrix} x\\ \xi \end{bmatrix} = \begin{bmatrix} A + B_2 D_k C_2 & B_2 C_k \\ B_k C_2 & A_k \end{bmatrix} \begin{bmatrix} x\\ \xi \end{bmatrix} + \begin{bmatrix} B_1 + B_2 D_k D_{21} \\ B_k D_{21} \end{bmatrix} w,$$
(7a)

$$z = \begin{bmatrix} C_1 + D_{12}D_kC_2 & D_{12}C_k \end{bmatrix} \begin{bmatrix} x\\ \xi \end{bmatrix} + (D_{11} + D_{12}D_kD_{21})w.$$
(7b)

This is a state-space version of the closed-loop response from w to z. We can write

$$\mathbf{T}_{zw} = \begin{bmatrix} A + B_2 D_k C_2 & B_2 C_k & B_1 + B_2 D_k D_{21} \\ B_k C_2 & A_k & B_k D_{21} \\ \hline C_1 + D_{12} D_k C_2 & D_{12} C_k & D_{11} + D_{12} D_k D_{21} \end{bmatrix}.$$

<sup>1</sup>This can be easily seen from the fact  $\begin{bmatrix} I & 0 \\ D_k & I \end{bmatrix} \begin{bmatrix} I & -D_{22} \\ -D_k & I \end{bmatrix} = \begin{bmatrix} I & -D_{22} \\ 0 & I - D_k D_{22} \end{bmatrix}.$ 

Consider the special case of static output feedback control  $u = D_k y$ . The closed-loop matrix is  $A + B_2 D_k C_2$ , and we have

$$\mathbf{T}_{zw} = \begin{bmatrix} A + B_2 D_k C_2 & B_1 + B_2 D_k D_{21} \\ \hline C_1 + D_{12} D_k C_2 & D_{11} + D_{12} D_k D_{21} \end{bmatrix}.$$

### **1.3** Internal stability

We now define the fundamental notion of internal stability.

**Definition 2.** The system in Fig. 1 is internally stable if it is well-posed, and the states  $(x(t), \xi(t))$  converge to zero as  $t \to \infty$  for all initial states  $x(0), \xi(0)$  when  $w(t) = 0, \forall t$ .

Lemma 2. The system in Fig 1 is internally stable if and only if

$$\hat{A} := \begin{bmatrix} A + B_2 D_k C_2 & B_2 C_k \\ B_k C_2 & A_k \end{bmatrix}$$

 $is\ stable.$ 

The set of all stabilizing controllers is defined as

$$\mathcal{C}_{\text{stab}} := \{ \mathbf{K} \mid \mathbf{K} \text{ internally stabilizes } \mathbf{P} \}.$$
(8)

It is well-known that  $C_{stab}$  is non-convex and it is not difficult to find explicit examples where  $\mathbf{K}_1, \mathbf{K}_2 \in C_{stab}$  and  $\frac{1}{2}(\mathbf{K}_1 + \mathbf{K}_2) \notin C_{stab}$ . Lemma 2 leads to an explicit state-space characterization of the set  $C_{stab}$  as follows:

$$\mathcal{C}_{\text{stab}} = \left\{ \mathbf{K} \mid \hat{A} := \begin{bmatrix} A + B_2 D_k C_2 & B_2 C_k \\ B_k C_2 & A_k \end{bmatrix} \text{ is stable} \right\},\tag{9}$$

where  $\mathbf{K} = C_k(zI - A_k)^{-1}B_k + D_k$ . Unfortunately, the stability condition on  $A_{cl}$  in (9) is still non-convex in terms of the parameters  $(A_k, B_k, C_k, D_k)$ .

# 2 Optimal control

Now, the optimal controller synthesis problem (3) can be precisely written as

$$\min_{\mathbf{K}} \|\mathbf{P}_{11} + \mathbf{P}_{12}\mathbf{K}(I - \mathbf{P}_{22}\mathbf{K})^{-1}\mathbf{P}_{21}\|$$
  
subject to  $\mathbf{K} \in \mathcal{C}_{\text{stab}}.$  (10)

The state-space version is

Both (10) and (11) are non-convex in its present form.

The rest of topics will include

- 1. Performance specification:  $\mathcal{H}_2$  and  $\mathcal{H}_\infty$  norms of transfer matrices and their computations via convex optimization (LMIs).
- 2. Convex reformulation of (10) in the frequency domain (Youla parameterization [9], system-level synthesis [1,8], and input-output parameterization [3,10]).
- 3. Convex reformulation of (11) in the state-space domain (convex optimization via LMIs) [6,7].
- 4. Analytical solutions via solving Algebraic Riccati Equation (ARE) [2,11].
- 5. Distributed control when introducing a subspace constraint on the controller  $\mathbf{K} \in \mathcal{S}$  (Quadratic Invariance [5], Sparsity Invariance [4], etc.).

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